

Beam Design - BAGS

1. Beam Data

2. Design Loads

Load Type: Support: Simple Beam Glulam Beam Type: Species: Western Species Grade: 24F-V4 1.8E DF/DF Size: 12.25 x 36 Design Span (L): 49.00 ft. Clear Span: 48.00 ft. 50.00 ft. Total Span: 12 in. Bearing (lb):

Uniform Dist. Load

Live Load: 975 plf Dead Load: 75 plf Selfweight: 5066.3 lbs Dist. Selfweight: 103.39 plf Total Weight: 5169.7 lbs

3. Design Options

Quantity (N):

4. Design Assumptions and Notes

Lateral Support: 8' o/c Defl. Limits: 360|240 Load Duration: 1.15 Exposure: dry $T \le 100$ °F Temperature: Vertical Orientation:

Code Standard: IBC 2015, NDS 2015 Bending Stress: Parallel to Grain

Notes:

5. Adjustment Factors

Factor	Description	Fb	Ft	$F_{\mathbf{v}}$	Fc	Fc⊥	E/E _{min}
C_{D}	Load Duration Factor	1.15	1.15	1.15	1.15	-	-
CM	Wet Service Factor	1	1	1	1	1	1
Ct	Temperature Factor	1	1	1	1	1	1
C_{L}	Beam Stability Factor	0.993	-	-	-	-	-
Cv	Volume Factor	0.754 ^b	-	-	_	-	-
Cfu	Flat Use Factor	N/A ^c	-	-	-	-	-

a) Adjustment factors per AWC NDS 2015 and NDS 2015 Supplement.

1

- b) The volume factor, C_V, shall not apply simultaneously with the beam stability factor, C_L. The lesser factor shall apply.
- c) Only applies when glulam beam is loaded in bending about the y-y axis.

Subject	Customer	Location			Job No.
Beam Design	Nick Bagatelos		Reno Nevada		2023A75
N. Wilkerson	MEDEEK ENGINE			This report may not be copied, reproduced or distributed without the written consent of Medek Engineering Inc.	Rev.
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	s Beach, WA 98535 edeek.com		Copyright © 2014	Page 1

6. Beam Calculations

Determine reference design values, sectional properties and self weight of beam:

$$A = b \times d$$

$$S_x = \frac{bd^2}{6}, \ S_y = \frac{b^2d}{6}$$

$$I_x = \frac{bd^3}{12}, \ I_y = \frac{b^3d}{12}$$

where:

b = Breadth of rectangular beam in bending (in.)

d = Depth of rectangular beam in bending (in.)

A = Cross sectional area of beam (in.²)

 S_x = Section modulus about the X-X axis (in.³)

 S_y = Section modulus about the Y-Y axis (in.³)

 I_X = Moment of inertia about the X-X axis (in.⁴)

 $I_{\rm V}$ = Moment of inertia about the Y-Y axis (in. 4)

b = 12.250 in.

d = 36.000 in.

$$A = 12.250 \times 36.000 = 441.00 \text{ in.}^2$$

$$S_x = (12.250)(36.000)^2/6 = 2646.00 \text{ in.}^3$$

$$S_V = (12.250)^2 (36.000)/6 = 900.38 \text{ in.}^3$$

$$I_x = (12.250)(36.000)^3/12 = 47628.00 \text{ in.}^4$$

$$I_V = (12.250)^3 (36.000)/12 = 5514.80 \text{ in.}^4$$

Reference Design Values from Table 5A NDS Supplement (Reference Design Values for Structural Glue Laminated Softwood Timber Combinations).

Species & Grade	F _{bx} +	F _{bx} -	$F_{c\perp x}$	F _{vx}	Ex	Eminx	Fby	$F_{c\perp y}$	Fvy	Ey	Eminy	Ft	Fc	G
24F-V4 1.8E DF/DF	2400	1850	650	265	1800000	950000	1450	560	230	1600000	850000	1100	1650	0.5

The following formula shall be used to determine the density of wood (lbs/ft³. (NDS Supplement Sec. 3.1.3)

$$\rho_w = 62.4 \left[\frac{G}{1 + G(0.009)(m.c)} \right] \left[1 + \frac{m.c.}{100} \right]$$

where:

 $\rho_{\rm W}$ = Density of wood (lbs/ft³

G = Specific gravity of wood (dimensionless)

m.c. = Moisture content of wood (percentile)

G = 0.5

m.c. = 16 % (Max. moisture content at dry service conditions)

Subject E	Beam Design	Customer Nick Bagatelos	Location	Reno Nevada		Job No. 2023A75
Engr.	N. Wilkerson	MEDEEK ENGINE			Rev.	
Date	3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.n	•		Medeek Engineering Inc. Copyright © 2014	Page 2

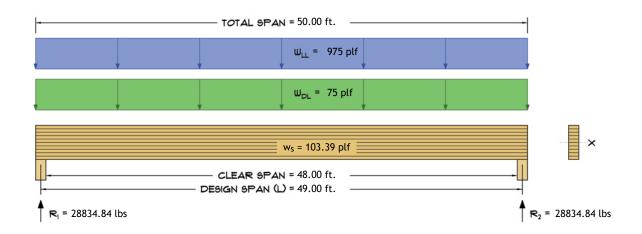
$$\rho_w = 62.4 \left[\frac{0.5}{1 + 0.5(0.009)(16)} \right] \left[1 + \frac{16}{100} \right] = 33.76 \text{ lbs/ft}^3$$

 $\begin{aligned} & Volume_{total} = N[A \ x \ (L + l_b)] = 1 \ x \ [441.00 \ x \ (588.00 + 12)] \ x \ (12 \ in./ft.)^3 = 153.13 \ ft^3 \\ & Volume_{span} = N[A \ x \ L] = 1 \ x \ [441.00 \ x \ 588.00] \ x \ (12 \ in./ft.)^3 = 150.06 \ ft^3 \end{aligned}$

Total Weight (W_T) = ρ_W x Volume_{total} = 33.76 x 153.13 = 5169.7 lbs Self Weight (W_S) = ρ_W x Volume_{span} = 33.76 x 150.06 = 5066.3 lbs

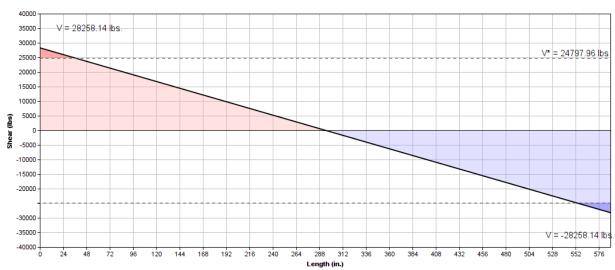
Distributed Self Weight (w_s) =
$$\frac{W_S}{L} = \frac{5066.3}{49.00} = 103.39 \text{ plf}$$

Load, Shear and Moment Diagrams:



Beam - Shear Diagram

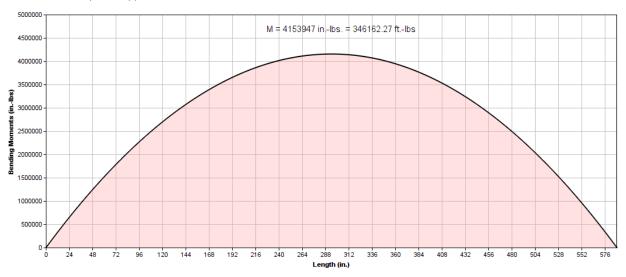
Shear Equation: V(x) = -96.12x + 28258.1



Subject	Customer	Location		Job No.
Beam Design	Nick Bagatelos	Reno Nevada		2023A75
N. Wilkerson	MEDEEK ENGINE		This report may not be copied, reproduced or distributed without the written consent of Medeek Engineering Inc.	Rev.
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	is Beach, WA 98535 nedeek.com	Copyright © 2014	Page 3

Beam - Moment Diagram

Moment Equation: M(x) = -48.06x2 + 28258.1x



1.) Bending:

Members subject to bending stresses shall be proportioned so that the actual bending stress or moment shall not exceed the adjusted bending design value:

$$f_b \le F_{b'}$$
 (NDS Sec. 3.3.1)

where:

$$f_b = M / S$$

$$F_{bx'} = F_{bx}(C_D)(C_M)(C_t)(C_V) \quad \text{or} \quad F_{bx'} = F_{bx}(C_D)(C_M)(C_t)(C_L)$$

Beam is unbraced along its compression edge, lateral stability is considered below:

Slenderness Ratio for bending member RB:

 $l_u = Unbraced Length = 8 ft.$

$$l_u/d = \frac{96}{36} = 2.67$$

$$l_e = 2.06l_u = 2.06(96.0) = 197.76$$
 in. = 16.48 ft. (NDS Table 3.3.3)

$$R_b = \sqrt{\frac{l_e d}{b^2}} = \sqrt{\frac{197.76(36)}{(1 \times 12.25)^2}} = 6.89$$

$$R_b = 6.89 < 50$$
 ? **OK**

Subject	Customer	Location	Job No.
Beam Design	Nick Bagatelos	Reno Nevada	2023A75
N. Wilkerson	MEDEEK ENGINE	written conse	d or the it of
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	lis Beach, WA 98535 nedeek.com	Page 4

Euler-based ASD critical buckling value for bending members:

$$E_{miny}' = E_{miny}(C_M)(C_t) = 850000(1)(1) = 850000 \text{ psi}$$

$$F_{bE} = \frac{1.2E'_{miny}}{(R_b)^2} = \frac{1.2(850000)}{(6.89)^2} = 21499.65 \text{ psi}$$

$$F_{bx}$$
* = $F_{bx}(C_D)(C_M)(C_t)$ = (2400)(1.15)(1)(1) = 2760.00 psi

Beam stability factor:

$$C_{L} = \frac{1 + F_{be}/F_{bx}^{*}}{1.9} - \sqrt{\left(\frac{1 + F_{be}/F_{bx}^{*}}{1.9}\right)^{2} - \frac{F_{be}/F_{bx}^{*}}{0.95}} = \frac{1 + 21499.65/2760.00}{1.9} - \sqrt{\left(\frac{1 + 21499.65/2760.00}{1.9}\right)^{2} - \frac{21499.65/2760.00}{0.95}} = \mathbf{0.993}$$

$$C_{V} = \left(\frac{21}{L}\right)^{0.1} \left(\frac{12}{d}\right)^{0.1} \left(\frac{5.125}{b}\right)^{0.1} = \left(\frac{21}{49.000}\right)^{0.1} \left(\frac{12}{36}\right)^{0.1} \left(\frac{5.125}{12.25}\right)^{0.1} = 0.754$$

Volume effect governs over lateral stability:

$$C_V = 0.754 < C_L = 0.993$$

$$F_{bx}' = (2400)(1.15)(1)(1)(0.754) = 2082.4 \text{ psi}$$

$$f_b = \frac{M}{N \times S_x} = \frac{4153947}{1 \times 2646.00} = 1569.9 \text{ psi}$$

$$f_b = 1569.9 \text{ psi} < F_{bx'} = 2082.4 \text{ psi} \text{ (CSI} = 0.75)$$
 ? **OK**

2.) Shear:

Members subject to shear stresses shall be proportioned so that the actual shear stress parallel to grain or shear force at any cross section of the bending member shall not exceed the adjusted shear design value:

$$f_V \le F_{V'}$$
 (NDS Sec. 3.4.1)

where:

$$\mathbf{f_v} = \frac{3V}{2A}$$

$$F_{v'} = F_{v}(C_{D})(C_{M})(C_{t})$$

$$F_{vx'} = (265)(1.15)(1)(1) = 304.75 \text{ psi}$$

Shear Reduction: Uniformly distributed loads within a distance, d, from supports equal to the depth of the bending member shall be permitted to be ignored. Concentrated loads within a distance equal to the depth of the bending member from supports shall be permitted to be multiplied by x/d where x is the distance from the beam support face to the load. See NDS 2015, Figure 3C.

$$\mathbf{f_{v}}^* = \frac{3V^*}{2(N \times A)} = \frac{3(24797.96)}{2(1 \times 441.00)} = 84.35 \text{ psi}$$

Beam Design	Customer Nick Bagatelos	Reno Nevada	Job No. 2023A75
N. Wilkerson	MEDEEK ENGINE	written co	uced or nout the nsent of
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	llis Beach, WA 98535	Page 5

$$f_v^* = 84.35 \text{ psi} < F_{vx'} = 304.75 \text{ psi} \text{ (CSI} = 0.28) ? OK$$

No Reduction in Shear (conservative):

$$\mathbf{f_V} = \frac{3V}{2(N \times A)} = \frac{3(28258.14)}{2(1 \times 441.00)} = 96.12 \text{ psi}$$

$$f_v = 96.12 \text{ psi} < F_{vx'} = 304.75 \text{ psi } (CSI = 0.32)$$
 ? **OK**

3.) Deflection:

Bending deflections calculated per standard method of engineering mechanics for live load and total load:

LL Allowable: L/360 TL Allowable: L/240

$$E_{x'} = E_{x}(C_{M})(C_{t}) = 1800000(1)(1) = 18000000 \text{ psi}$$

$$\Delta_{\rm LL} = \frac{5w_{LL}L^4}{384E_x'(N\times I_x)} = \frac{5(975)(49.000)^4}{384(1800000)(1\times47628.00)} \times \left(12\frac{in.}{ft.}\right)^3 = 1.48 \text{ in.}$$

$$(L/d)_{LL} = 588.00 / 1.48 = 399$$

$$\Delta_{LL} = 1.48 \text{ in} = L/399 < L/360 ? OK$$

$$\Delta_{\rm TL} = \frac{5(w_{TL} + w_s)L^4}{384E_x'(N \times I_x)} = \frac{5(1050 + 103.39)(49.000)^4}{384(1800000)(1 \times 47628.00)} \times \left(12\frac{in.}{ft.}\right)^3 = 1.75 \text{ in.}$$

$$(L/d)_{TL} = 588.00 / 1.75 = 337$$

$$\Delta_{TL} = 1.75 \text{ in} = L/337 < L/240$$
 ? **OK**

4.) Bearing:

Members subject to bearing stresses perpendicular to the grain shall be proportioned so that the actual compressive stress perpendicular to grain shall be based on the net bearing area and shall not exceed the adjusted compression design value perpendicular to grain:

$$f_{c\perp} \leq F_{c\perp}$$
' (NDS Sec. 3.10.2)

where:

$$f_{c\perp} = \frac{R}{A_b}$$

$$F_{c\perp}' = F_{c\perp}(C_M)(C_t)$$

$$F_{c \perp x'} = (650)(1)(1) = 650.00 \text{ psi}$$

Subject	Customer	Location	Job No.
Beam Design	Nick Bagatelos	Reno Nevada	2023A75
Engr. N. Wilkerson	MEDEEK ENGINE	This report may not be copied, reproduced or distributed without the written consent of Medeek Engineering Inc.	Rev.
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	ilis Beach, WA 98535	Page 6

$$A_b = b \times l_b = 12.25 \times 12 = 147.00 \text{ in}^2$$

$$\mathbf{f_{c}}_{\perp} = \frac{R}{N \times A_b} = \frac{28834.84}{1 \times 147.00} = 196.2 \; \mathrm{psi}$$

$$f_{c\,\perp} = 196.2~psi < F_{c\,\perp\,x}{}^{\prime} = 650.00~psi~(CSI = 0.30)~?$$

 OK

*Disclaimer: The calculations produced herein are for initial design and estimating purposes only. The calculations and drawings presented do not constitute a fully engineered design. All of the potential load cases required to fully design an actual structure may not be provided by this calculator. For the design of an actual structure, a registered and licensed professional should be consulted as per IRC 2012 Sec. R802.10.2 and designed according to the minimum requirements of ASCE 7-10. The beam calculations provided by this online tool are for educational and illustrative purposes only. Medeek Design assumes no liability or loss for any designs presented and does not guarantee fitness for use.

Subject	Customer	Location			Job No.
Beam Design	Nick Bagatelos		Reno Nevada		2023A75
N. Wilkerson	MEDEEK ENGINEERING INC.		This report may not be copied, reproduced or distributed without the written consent of Medeek Engineering Inc.		Rev.
3/14/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	·		Copyright © 2014	Page 7