

Beam Design - Melissa Hunt Basement beam

1. Beam Data

Uniform Dist. Load Live Load: Load Type: 640 plf Support: Simple Beam Dead Load: Sawn Lumber Beam Type: Selfweight: Spruce-Pine-Fir Species: SPF SS Grade:

Size: 2 x 12 Design Span (L): 11.75 ft. 11.50 ft. Clear Span: 12.00 ft. Total Span: Bearing (lb): 3 in. 4 Quantity (N):

3. Design Options

Lateral Support: braced Defl. Limits: 180|120 Load Duration: 1.15 Exposure: dry $T \le 100^{\circ}F$ Temperature: Orientation: Vertical Incised Lumber: No Rep. Members: No

2. Design Loads

240 plf 160.3 lbs Dist. Selfweight: 13.64 plf Total Weight: 163.7 lbs

4. Design Assumptions and Notes

Code Standard: IBC 2015, NDS 2015 Bending Stress: Parallel to Grain Notes: Nelson Plan HR1319

5. Adjustment Factors

Factor	Description	Fb	Ft	$F_{\mathbf{v}}$	Fc	Fc⊥	E/E _{min}
C_{D}	Load Duration Factor	1.15	1.15	1.15	1.15	-	-
$C_{\mathbf{M}}$	Wet Service Factor	1 ^b	1	1	1 ^c	1	1
Ct	Temperature Factor	1	1	1	1	1	1
C_{L}	Beam Stability Factor	1	-	-	-	-	_
C_{F}	Size Factor	1	1	-	1	-	-
C_{fu}	Flat Use Factor	1.2 ^d	_	_	-	_	_
Ci	Incising Factor	1	1	1	1	1	1
Cr	Repetitive Member Factor	1	-	-	-	-	-

- a) Adjustment factors per AWC NDS 2015 and NDS 2015 Supplement.
- b) When $(F_b)(C_F) \le 1,150 \text{ psi}$, $C_M = 1.0$.
- c) When $(F_c)(C_F) \le 750 \text{ psi}$, $C_M = 1.0$.
- d) Only applies when sawn lumber or glulam beams are loaded in bending about the y-y axis.

Subject	Customer	Location	Job No.
Beam Design Ryan Bailey		PO Box 16, Station A, Calgary, AB	220728
N. Wilkerson	MEDEEK ENGINE	written consent of	Rev.
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	alis Beach, WA 98535 medeek.com	Page 1

6. Beam Calculations

Determine reference design values, sectional properties and self weight of beam:

 $A = b \times d$

$$S_x = \frac{bd^2}{6}, \ S_y = \frac{b^2d}{6}$$

$$I_x = \frac{bd^3}{12}, \ I_y = \frac{b^3d}{12}$$

where:

b = Breadth of rectangular beam in bending (in.)

d = Depth of rectangular beam in bending (in.)

A = Cross sectional area of beam (in.²)

 S_X = Section modulus about the X-X axis (in.³)

 S_y = Section modulus about the Y-Y axis (in.³)

 I_X = Moment of inertia about the X-X axis (in. 4)

 $I_y = Moment of inertia about the Y-Y axis (in.⁴)$

b = 1.500 in.

d = 11.250 in.

$$A = 1.500 \text{ x } 11.250 = 16.88 \text{ in.}^2$$

$$S_x = (1.500)(11.250)^2/6 = 31.64 \text{ in.}^3$$

$$S_V = (1.500)^2 (11.250)/6 = 4.22 \text{ in.}^3$$

$$I_x = (1.500)(11.250)^3/12 = 177.98 \text{ in.}^4$$

$$I_v = (1.500)^3 (11.250)/12 = 3.16 \text{ in.}^4$$

Reference Design Values from Table 4A NDS Supplement (Reference Design Values for Visually Graded Dimension Lumber, 2" - 4" thick).

Species & Grade	Fb	Ft	$F_{\mathbf{v}}$	Fc⊥	Fc	Е	Emin	G
SPF SS	1250	700	135	425	1400	1500000	550000	0.42

The following formula shall be used to determine the density of wood (lbs/ft³. (NDS Supplement Sec. 3.1.3)

$$\rho_w = 62.4 \left[\frac{G}{1 + G(0.009)(m.c)} \right] \left[1 + \frac{m.c.}{100} \right]$$

where:

 $\rho_{\rm W}$ = Density of wood (lbs/ft³

G = Specific gravity of wood (dimensionless)

m.c. = Moisture content of wood (percentile)

G = 0.42

m.c. = 19 % (Max. moisture content at dry service conditions)

Beam Design	Customer Ryan Bailey	PO Box 16, Station A, Calgary, AB	3
Engr. N. Wilkerson	MEDEEK ENGINE	written consent of	
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r		

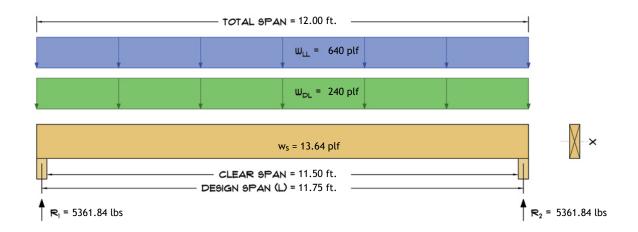
$$\rho_w = 62.4 \left\lceil \frac{0.42}{1 + 0.42(0.009)(19)} \right\rceil \left\lceil 1 + \frac{19}{100} \right\rceil = 29.10 \text{ lbs/ft}^3$$

Volume_{total} = N[A x (L + l_b)] = 4 x [16.88 x (141.00 + 3)] x (12 in./ft.)³ = 5.63 ft³ Volume_{span} = N[A x L] = 4 x [16.88 x 141.00] x $(12 \text{ in./ft.})^3 = 5.51 \text{ ft}^3$

Total Weight (W_T) = ρ_w x Volume_{total} = 29.10 x 5.63 = 163.7 lbs Self Weight (W_S) = ρ_{W} x Volume_{span} = 29.10 x 5.51 = 160.3 lbs

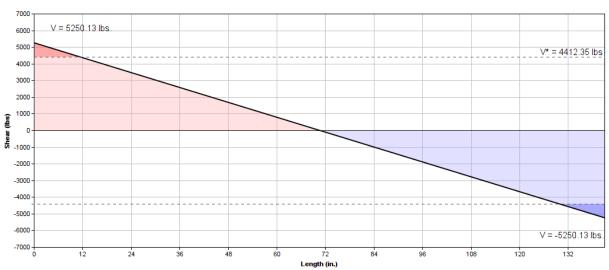
Distributed Self Weight (w_s) =
$$\frac{W_S}{L} = \frac{160.3}{11.75}$$
 = 13.64 plf

Load, Shear and Moment Diagrams:



Beam - Shear Diagram

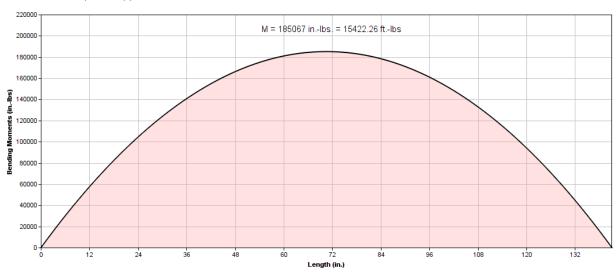
Shear Equation: V(x) = -74.47x + 5250.1



Subject Beam Design	Ryan Bailey PO Box 16, Station A, Calgary, AB		
Engr.	y y		Rev.
N. Wilkerson	MEDEEK ENGINE	written consent of	-
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.n	llis Beach, WA 98535	Page 3

Beam - Moment Diagram

Moment Equation: $M(x) = -37.23x^2 + 5250.1x$



1.) Bending:

Members subject to bending stresses shall be proportioned so that the actual bending stress or moment shall not exceed the adjusted bending design value:

$$f_b \le F_b$$
' (NDS Sec. 3.3.1)

where:

$$f_b = M / S$$

$$F_b' = F_b(C_D)(C_M)(C_t)(C_L)(C_F)(C_i)(C_r)$$

Beam is braced laterally along its compression edge. Laterial stability is not a consideration:

 C_L = Beam Stability Factor = 1.0

$$F_{bx}' = (1250)(1.15)(1)(1)(1)(1)(1)(1) = 1437.5 \text{ psi}$$

$${\rm fb} = \frac{M}{N \times S_x} = \frac{185067}{4 \times 31.64} = 1462.3 \ {\rm psi}$$

$$f_b = 1462.3 \text{ psi} > F_{bx'} = 1437.5 \text{ psi (CSI} = 1.02)$$
 ? **NG**

Subject	Customer	Location	Job No.	
Beam Design Ryan Bailey		PO Box 16, Station A, Calgary, AB	220728	
N. Wilkerson	MEDEEK ENGINEERING INC. This report may not be copied, reproduced o distributed without the written consent of written consent of the copied copied in the copied of the copied copied in the copied of the copied in the copied copied in the			
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r		Page 4	

2.) Shear:

Members subject to shear stresses shall be proportioned so that the actual shear stress parallel to grain or shear force at any cross section of the bending member shall not exceed the adjusted shear design value:

$$f_{v} \leq F_{v} \text{' (NDS Sec. 3.4.1)}$$

where:

$$\mathbf{f_v} = \frac{3V}{2A}$$

$$F_{v'} = F_{v}(C_D)(C_M)(C_t)(C_i)$$

$$F_{vx'} = (135)(1.15)(1)(1)(1) = 155.25 \text{ psi}$$

Shear Reduction: For beams supported by full bearing on one surface and loads applied to the opposite surface, uniformly distributed loads within a distance, d, from supports equal to the depth of the bending member shall be pemitted to be ignored. For beams supported by full bearing on one surface and loads applied to the opposite surface, concentrated loads within a distance equal to the depth of the bending member from supports shall be permitted to be multiplied by x/d where x is the distance from the beam support face to the load. See NDS 2015, Figure 3C.

$$\mathbf{f_{v}}^* = \frac{3V^*}{2(N \times A)} = \frac{3(4412.35)}{2(4 \times 16.88)} = 98.05 \text{ psi}$$

$$f_v^* = 98.05 \text{ psi} < F_{vx'} = 155.25 \text{ psi} \text{ (CSI} = 0.63) ? OK$$

No Reduction in Shear (conservative):

$$\mathbf{f_v} = \frac{3V}{2(N \times A)} = \frac{3(5250.13)}{2(4 \times 16.88)} = 116.67 \text{ psi}$$

$$f_v = 116.67 \text{ psi} < F_{vx'} = 155.25 \text{ psi} \text{ (CSI} = 0.75)$$
 ? **OK**

3.) Deflection:

Bending deflections calculated per standard method of engineering mechanics for live load and total load:

LL Allowable: L/180 TL Allowable: L/120

$$E_{x'} = E_{x}(C_{M})(C_{t})(C_{i}) = 1500000(1)(1)(1) = 1500000 \text{ psi}$$

Subject	Customer	Location	Job No.
Beam Design Ryan Bailey		PO Box 16, Station A, Calgary, AB	220728
N. Wilkerson	MEDEEK ENGINE	written conso	ed or t the nt of
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	alis Beach, WA 98535 medeek.com	Page 5

$$\Delta_{\rm LL} = \frac{5w_{LL}L^4}{384E_x'(N\times I_x)} = \frac{5(640)(11.750)^4}{384(1500000)(4\times177.98)}\times \left(12\frac{in.}{ft.}\right)^3 = 0.26 \ {\rm in}.$$

$$(L/d)_{LL} = 141.00 / 0.26 = 549$$

$$\Delta_{LL} = 0.26 \text{ in} = L/549 < L/180$$
 ? **OK**

$$\Delta_{\rm TL} = \frac{5(w_{TL} + w_s)L^4}{384 E_x'(N \times I_x)} = \frac{5(880 + 13.64)(11.750)^4}{384(1500000)(4 \times 177.98)} \times \left(12 \frac{in.}{ft.}\right)^3 = 0.36 \ {\rm in.}$$

$$(L/d)_{TL} = 141.00 / 0.36 = 393$$

$$\Delta_{TL} = 0.36 \text{ in} = L/393 < L/120$$
 ? **OK**

4.) Bearing:

Members subject to bearing stresses perpendicular to the grain shall be proportioned so that the actual compressive stress perpendicular to grain shall be based on the net bearing area and shall not exceed the adjusted compression design value perpendicular to grain:

$$f_{c\perp} \leq F_{c\perp}$$
' (NDS Sec. 3.10.2)

where:

$$f_{c\perp} = \frac{R}{A_b}$$

$$F_{c\perp}' = F_{c\perp}(C_M)(C_t)(C_i)$$

$$F_{c+x}' = (425)(1)(1)(1) = 425.00 \text{ psi}$$

$$A_b = b \times l_b = 1.5 \times 3 = 4.50 \text{ in}^2$$

$$\mathbf{f_{c\perp}} = \frac{R}{N \times A_b} = \frac{5361.84}{4 \times 4.50} = 297.9 \text{ psi}$$

$$f_{c\,\perp} = 297.9~psi < F_{c\,\perp\,x'} = 425.00~psi~(CSI = 0.70)~?$$
 OK

*Disclaimer: The calculations produced herein are for initial design and estimating purposes only. The calculations and drawings presented do not constitute a fully engineered design. All of the potential load cases required to fully design an actual structure may not be provided by this calculator. For the design of an actual structure, a registered and licensed professional should be consulted as per IRC 2012 Sec. R802.10.2 and designed according to the minimum requirements of ASCE 7-10. The beam calculations provided by this online tool are for educational and illustrative purposes only. Medeek Design assumes no liability or loss for any designs presented and does not guarantee fitness for use.

Beam Design	Ryan Bailey Customer PO Box		O Box 16, Station A, Calgary, AB		
N. Wilkerson	MEDEEK ENGINE			This report may not be copied, reproduced or distributed without the written consent of Medeek Engineering Inc.	Rev.
7/28/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	is Beach, WA 98535 nedeek.com		Copyright © 2014	Page 6