Beam Design - Header above Garage Door Check

1. Beam Data

Load Type:	Uniform Dist. Load
Support:	Simple Beam
Beam Type:	Sawn Lumber
Species:	Spruce-Pine-Fir
Grade:	SPF No.2
Size:	2 x 12
Design Span (L):	12.38 ft.
Clear Span:	12.00 ft.
Total Span:	12.75 ft.
Bearing (lb):	4.5 in.
Quantity (N):	3

2.	Design	Loads
	•	

Live Load:	227	plf
Dead Load:	156	plf
Selfweight:	126.6	lbs
Dist. Selfweight:	10.23	plf
Total Weight:	130.4	lbs

3. Design Options

Lateral Support:	braced
Defl. Limits:	360 240
Load Duration:	1.15
Exposure:	dry
Temperature:	$T \leq 100^{\circ}F$
Orientation:	Vertical
T ' 1T 1	N
Incised Lumber:	No

4. Design Assumptions and Notes

Code Standard:	IBC 2015, NDS 2015
Bending Stress:	Parallel to Grain
Notes:	Check on as Modified Header.

5. Adjustment Factors

Date

1/25/2023

	Factor	Description]	Fb	Ft	Fv	Fc	$F_{c\perp}$	E/E _{min}]
	CD	Load Duration Factor	or 1	.15	1.15	1.15	1.15	-	-	
	CM	Wet Service Factor	•	1 ^b	1	1	1 ^c	1	1	
	Ct	Temperature Factor	r	1	1	1	1	1	1	
	CL	Beam Stability Factor	or	1	-	-	-	-	-	1
	CF	Size Factor		1	1	-	1	-	-	
	C _{fu}	Flat Use Factor	1	.2 ^d	-	-	-	-	-	
	Ci	Incising Factor		1	1	1	1	1	1	
	Cr	Repetitive Member Fa	ctor	1	-	-	-	-	-	
	a) Adjust	ment factors per AWC NDS 2015 a	nd NDS 2015 Su	pplem	ent.					
	b) When	$(F_b)(C_F) \le 1,150 \text{ psi}, C_M = 1.0.$								
	c) When	$(F_c)(C_F) \le 750 \text{ psi}, C_M = 1.0.$								
	d) Only a	upplies when sawn lumber or glulam	beams are loade	ed in be	ending abou	t the y-y ax	s.			
Subject		Customer	Location							Job No.
Beam D	esign	Bouma Design			1166 V	idal Stree	t, Sarnia			230129
^{Engr.} N. Wilk	erson	MEDEEK ENGINE					KO	copi distri	report may not be ied, reproduced or ibuted without the written consent of k Engineering Inc.	Rev. _
Date		3050 State Route 109 Cons	alle Reach M/A	48534	h					Page

3050 State Route 109 Copalis Beach, WA 98535 ph. (425) 741-5555 www.medeek.com



Page

Copyright © 2014

1

6. Beam Calculations

Determine reference design values, sectional properties and self weight of beam:

$$A = b x d$$

$$S_x = \frac{bd^2}{6}, \ S_y = \frac{b^2d}{6}$$

 $I_x = \frac{bd^3}{12}, \ I_y = \frac{b^3d}{12}$

where:

b = Breadth of rectangular beam in bending (in.) d = Depth of rectangular beam in bending (in.) A = Cross sectional area of beam (in.²) S_x = Section modulus about the X-X axis (in.³) S_y = Section modulus about the Y-Y axis (in.³) I_x = Moment of inertia about the X-X axis (in.⁴) I_y = Moment of inertia about the Y-Y axis (in.⁴)

$$\begin{split} &b = 1.500 \text{ in.} \\ &d = 11.250 \text{ in.} \\ &A = 1.500 \text{ x } 11.250 = 16.88 \text{ in.}^2 \\ &S_x = (1.500)(11.250)^2/6 = 31.64 \text{ in.}^3 \\ &S_y = (1.500)^2(11.250)/6 = 4.22 \text{ in.}^3 \\ &I_x = (1.500)(11.250)^3/12 = 177.98 \text{ in.}^4 \\ &I_y = (1.500)^3(11.250)/12 = 3.16 \text{ in.}^4 \end{split}$$

Reference Design Values from Table 4A NDS Supplement (Reference Design Values for Visually Graded Dimension Lumber, 2" - 4" thick).

Specie	es & Grade	Fb	Ft	Fv	$F_{c\perp}$	Fc	E	Emin	G
SP	F No.2	875	450	135	425	1150	1400000	510000	0.42

The following formula shall be used to determine the density of wood (lbs/ft³. (NDS Supplement Sec. 3.1.3)

$$\rho_w = 62.4 \left[\frac{G}{1 + G(0.009)(m.c)} \right] \left[1 + \frac{m.c.}{100} \right]$$

where:

 ρ_w = Density of wood (lbs/ft³) G = Specific gravity of wood (dimensionless) m.c. = Moisture content of wood (percentile)

G = 0.42m.c. = 19 % (Max. moisture content at dry service conditions)

Beam Design	Customer Bouma Design	Location 1166 Vidal Street, Sarnia	Job No. 230129
^{Engr.} N. Wilkerson	MEDEEK ENGINE	written consent of Modewise Engineering Inc.	Rev. -
Date 1/25/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	lis Beach, WA 98535	Page 2

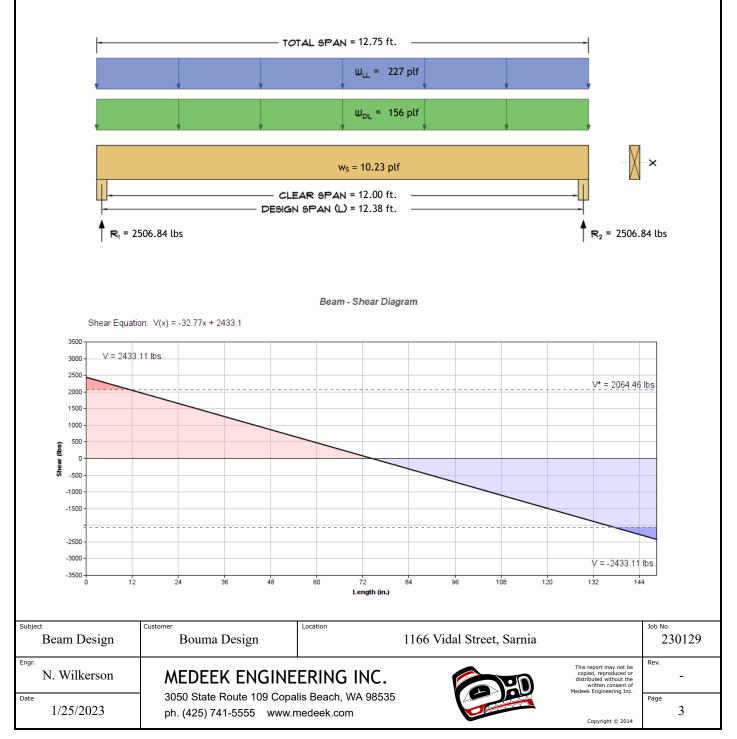
$$\rho_w = 62.4 \left[\frac{0.42}{1 + 0.42(0.009)(19)} \right] \left[1 + \frac{19}{100} \right] = 29.10 \text{ lbs/ft}^3$$

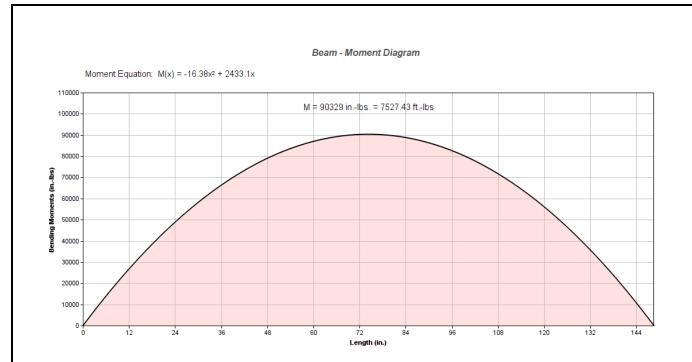
 $\begin{aligned} &\text{Volume}_{total} = \text{N}[\text{A x } (\text{L} + \text{l}_{b})] = 3 \text{ x } [16.88 \text{ x } (148.50 + 4.5)] \text{ x } (12 \text{ in./ft.})^{3} = 4.48 \text{ ft}^{3} \\ &\text{Volume}_{span} = \text{N}[\text{A x } \text{L}] = 3 \text{ x } [16.88 \text{ x } 148.50] \text{ x } (12 \text{ in./ft.})^{3} = 4.35 \text{ ft}^{3} \end{aligned}$

Total Weight (W_T) = $\rho_W x$ Volume_{total} = 29.10 x 4.48 = 130.4 lbs Self Weight (W_S) = $\rho_W x$ Volume_{span} = 29.10 x 4.35 = 126.6 lbs

Distributed Self Weight (w_s) = $\frac{W_S}{L} = \frac{126.6}{12.38}$ = 10.23 plf

Load, Shear and Moment Diagrams:





1.) Bending:

Members subject to bending stresses shall be proportioned so that the actual bending stress or moment shall not exceed the adjusted bending design value:

 $f_b \leq F_b' \ (\textit{NDS Sec. 3.3.1})$

where:

$$\label{eq:fb} \begin{split} f_b &= M \ / \ S \\ F_b' &= F_b(C_D)(C_M)(C_t)(C_L)(C_F)(C_i)(C_r) \end{split}$$

Beam is braced laterally along its compression edge. Laterial stability is not a consideration:

 C_L = Beam Stability Factor = 1.0

 $F_{bx'} = (875)(1.15)(1)(1)(1)(1)(1)(1) = 1006.2 \text{ psi}$

$$f_b = \frac{M}{N \times S_x} = \frac{90329}{3 \times 31.64} = 951.6 \text{ psi}$$

$$f_b = 951.6 \text{ psi} < F_{bx'} = 1006.2 \text{ psi} (CSI = 0.95)$$
 ? **OK**

Subject Beam Design	Customer Bouma Design	Location 1166 Vidal Street, Sarnia	Job No. 230129
Dealli Desigli	Douma Design	1100 Vida Street, Saina	250125
^{Engr.} N. Wilkerson	MEDEEK ENGINE	written conse	for the t of
Date 1/25/2023	3050 State Route 109 Copal ph. (425) 741-5555 www.m	lis Beach, WA 98535	Page 4

2.) Shear:

Members subject to shear stresses shall be proportioned so that the actual shear stress parallel to grain or shear force at any cross section of the bending member shall not exceed the adjusted shear design value:

 $f_V \leq F_V'$ (NDS Sec. 3.4.1)

where:

$$\mathbf{f_v} = \frac{3V}{2A}$$

 $F_{v}' = F_{v}(C_{D})(C_{M})(C_{t})(C_{i})$

$$F_{vx'} = (135)(1.15)(1)(1)(1) = 155.25 \text{ psi}$$

Shear Reduction: For beams supported by full bearing on one surface and loads applied to the opposite surface, uniformly distributed loads within a distance, d, from supports equal to the depth of the bending member shall be pemitted to be ignored. For beams supported by full bearing on one surface and loads applied to the opposite surface, concentrated loads within a distance equal to the depth of the bending member shall be permitted to be multiplied by x/d where x is the distance from the beam support face to the load. See NDS 2015, Figure 3C.

$$\mathbf{f_V}^* = \frac{3V^*}{2(N \times A)} = \frac{3(2064.46)}{2(3 \times 16.88)} = 61.17 \text{ psi}$$

$$f_v^* = 61.17 \text{ psi} < F_{vx'} = 155.25 \text{ psi} (CSI = 0.39)$$
 ? **OK**

No Reduction in Shear (conservative):

$$f_v = \frac{3V}{2(N \times A)} = \frac{3(2433.11)}{2(3 \times 16.88)} = 72.09 \text{ psi}$$

 $f_v = 72.09 \; psi < F_{vx}{'} = 155.25 \; psi \; \; (CSI = 0.46) \; \; ? \; \mbox{OK}$

3.) Deflection:

Bending deflections calculated per standard method of engineering mechanics for live load and total load:

LL Allowable: L/360 TL Allowable: L/240

 $E_x' = E_x(C_M)(C_t)(C_i) = 1400000(1)(1)(1) = 1400000 \text{ psi}$

Subject	Customer	Location		Job No.
Beam Design	Bouma Design	1166 Vidal Street, Sarnia		230129
^{Engr.} N. Wilkerson	MEDEEK ENGINEI		This report may not be copied, reproduced or distributed without the written consent of Medeek Engineering Inc.	Rev. -
Date 1/25/2023	3050 State Route 109 Copali ph. (425) 741-5555 www.m		Copyright © 2014	Page 5

$$\Delta_{LL} = \frac{5w_{LL}L^4}{384E'_x(N \times I_x)} = \frac{5(227)(12.375)^4}{384(1400000)(3 \times 177.98)} \times \left(12\frac{in.}{ft.}\right)^3 = 0.16 \text{ in.}$$

$$(L/d)_{LL} = 148.50 / 0.16 = 927$$

$$\Delta_{LL} = 0.16 \text{ in} = L/927 < L/360 ? \text{OK}$$

$$\Delta_{TL} = \frac{5(w_{TL} + w_s)L^4}{384E'_x(N \times I_x)} = \frac{5(383 + 10.23)(12.375)^4}{384(1400000)(3 \times 177.98)} \times \left(12\frac{in.}{ft.}\right)^3 = 0.28 \text{ in.}$$

$$(L/d)_{TL} = 148.50 / 0.28 = 535$$

$$\Delta_{TL} = 0.28 \text{ in} = L/535 < L/240 ? \text{OK}$$

4.) Bearing:

Members subject to bearing stresses perpendicular to the grain shall be proportioned so that the actual compressive stress perpendicular to grain shall be based on the net bearing area and shall not exceed the adjusted compression design value perpendicular to grain:

 $f_{c\perp} \leq F_{c\perp}$ ' (NDS Sec. 3.10.2)

where:

$$\mathbf{f_{c\perp}} = \frac{R}{A_b}$$

 $F_{c\perp}' = F_{c\perp}(C_M)(C_t)(C_i)$

 $F_{c \perp x'} = (425)(1)(1)(1) = 425.00 \text{ psi}$

 $A_b = b x l_b = 1.5 x 4.5 = 6.75 in^2$

 ${f f_c}_\perp = rac{R}{N imes A_b} = rac{2506.84}{3 imes 6.75} = 123.8 \; {
m psi}$

 $f_{c\,\perp} = 123.8 \; psi < F_{c\,\perp\,x'} = 425.00 \; psi \; (CSI = 0.29) \; ?$ OK

*Disclaimer: The calculations produced herein are for initial design and estimating purposes only. The calculations and drawings presented do not constitute a fully engineered design. All of the potential load cases required to fully design an actual structure may not be provided by this calculator. For the design of an actual structure, a registered and licensed professional should be consulted as per IRC 2012 Sec. R802.10.2 and designed according to the minimum requirements of ASCE 7-10. The beam calculations provided by this online tool are for educational and illustrative purposes only. Medeek Design assumes no liability or loss for any designs presented and does not guarantee fitness for use.

Subject	Customer	tion Job No.	
Beam Design	Bouma Design	1166 Vidal Street, Sarnia 2	30129
^{Engr.} N. Wilkerson	MEDEEK ENGINE	written consent of	-
Date 1/25/2023	3050 State Route 109 Copa ph. (425) 741-5555 www.r	each, WA 98535 eek.com Copyright © 2014	6