Beam Design - Beam

1. Beam Data

2.	Design	Loads
	<u> </u>	

Load Type:	Uniform Dist. Load	Live Load:
Support:	Simple Beam	Dead Load:
Beam Type:	Glulam	Selfweight:
Species:	Western Species	Dist. Selfweig
Grade:	24F-V4 1.8E DF/DF	Total Weight:
Size:	2.5 x 16.5	
Design Span (L):	14.67 ft.	
Clear Span:	14.33 ft.	
Total Span:	15.00 ft.	
Bearing (lb):	4 in.	
Quantity (N):	1	

Live Load:	450	plf
Dead Load:	337	plf
Selfweight:	141.8	lbs
Dist. Selfweight:	9.67	plf
Total Weight:	145.1	lbs

3. Design Options

Lateral Support:	braced
Defl. Limits:	360 240
Load Duration:	1.33
Exposure:	dry
Temperature:	$T \leq 100^{\circ}F$
Orientation:	Vertical

4. Design Assumptions and Notes

Code Standard:	IBC 2015, NDS 2015
Bending Stress:	Parallel to Grain
Notes:	

5. Adjustment Factors

Factor	Description	Fb	Ft	Fv	Fc	$F_{c\perp}$	E/E _{min}
CD	Load Duration Factor	1.33	1.33	1.33	1.33	-	-
CM	Wet Service Factor	1	1	1	1	1	1
Ct	Temperature Factor	1	1	1	1	1	1
CL	Beam Stability Factor	1	-	-	-	-	-
Cv	Volume Factor	1.0 ^b	-	-	-	-	-
C _{fu}	Flat Use Factor	N/A ^c	-	-	-	-	-

a) Adjustment factors per AWC NDS 2015 and NDS 2015 Supplement.

b) The volume factor, CV, shall not apply simultaneously with the beam stability factor, CL. The lesser factor shall apply.

c) Only applies when glulam beam is loaded in bending about the y-y axis.

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6. Beam Calculations

Determine reference design values, sectional properties and self weight of beam:

A = b x d

,

,

where:

b = Breadth of rectangular beam in bending (in.)

d = Depth of rectangular beam in bending (in.)

A = Cross sectional area of beam $(in.^2)$

 S_x = Section modulus about the X-X axis (in.³)

 S_y = Section modulus about the Y-Y axis (in.³)

 I_x = Moment of inertia about the X-X axis (in.⁴)

 I_y = Moment of inertia about the Y-Y axis (in.⁴)

$$\begin{split} &b = 2.500 \text{ in.} \\ &d = 16.500 \text{ in.} \\ &A = 2.500 \text{ x } 16.500 = 41.25 \text{ in.}^2 \\ &S_x = (2.500)(16.500)^2/6 = 113.44 \text{ in.}^3 \\ &S_y = (2.500)^2(16.500)/6 = 17.19 \text{ in.}^3 \\ &I_x = (2.500)(16.500)^3/12 = 935.86 \text{ in.}^4 \\ &I_y = (2.500)^3(16.500)/12 = 21.48 \text{ in.}^4 \end{split}$$

Reference Design Values from Table 5A NDS Supplement (Reference Design Values for Structural Glue Laminated Softwood Timber Combinations).

Species & Grade	Fbx+	Fbx-	$F_{c\perpx}$	Fvx	Ex	Eminx	Fby	$F_{c\perpy}$	Fvy	Ey	Eminy	Ft	Fc	G
24F-V4 1.8E DF/DF	2400	1850	650	265	1800000	950000	1450	560	230	1600000	850000	1100	1650	0.5

The following formula shall be used to determine the density of wood (lbs/ft³. (NDS Supplement Sec. 3.1.3)

where:

 ρ_w = Density of wood (lbs/ft³) G = Specific gravity of wood (dimensionless) m.c. = Moisture content of wood (percentile)

G = 0.5

m.c. = 16 % (Max. moisture content at dry service conditions)

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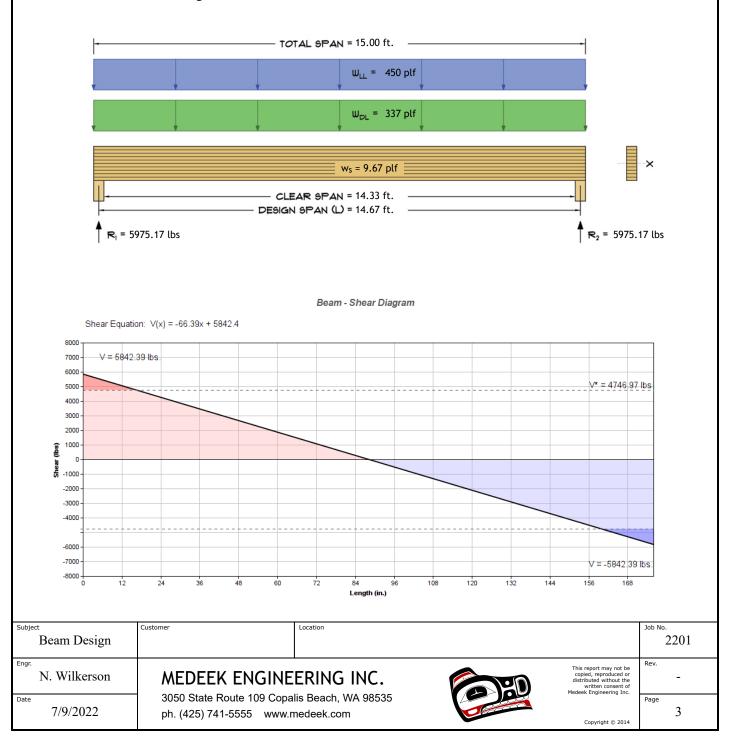
$= 33.76 \text{ lbs/ft}^3$

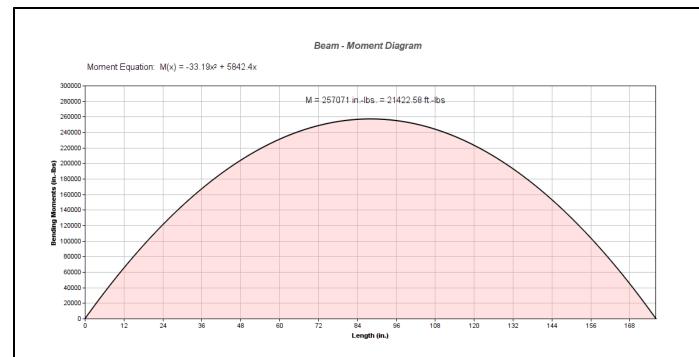
Volume_{total} = N[A x (L + l_b)] = 1 x [41.25 x (176.00 + 4)] x (12 in./ft.)³ = 4.30 ft³ Volume_{span} = N[A x L] = 1 x [41.25 x 176.00] x (12 in./ft.)³ = 4.20 ft³

Total Weight (W_T) = $\rho_W x$ Volume_{total} = 33.76 x 4.30 = 145.1 lbs Self Weight (W_S) = $\rho_W x$ Volume_{span} = 33.76 x 4.20 = 141.8 lbs

Distributed Self Weight $(w_s) = = 9.67 \text{ plf}$

Load, Shear and Moment Diagrams:





1.) Bending:

Members subject to bending stresses shall be proportioned so that the actual bending stress or moment shall not exceed the adjusted bending design value:

 $f_b \leq F_b' ~(\textit{NDS Sec. 3.3.1})$

where:

$$\begin{aligned} f_b &= M / S \\ F_{bx}' &= F_{bx}(C_D)(C_M)(C_t)(C_V) \quad \text{or} \quad F_{bx}' &= F_{bx}(C_D)(C_M)(C_t)(C_L) \end{aligned}$$

Beam is braced laterally along its compression edge. Laterial stability is not a consideration:

 C_L = Beam Stability Factor = 1.0

 $C_{V} = = 1.0$

Neither volume effect nor lateral stability govern:

 $C_{V} = 1.0$

 $F_{bx}' = (2400)(1.33)(1)(1)(1.0) = 3192.0 \text{ psi}$

 $f_b = = 2266.2 \text{ psi}$

 $f_b = 2266.2 \ psi < F_{bx}' = 3192.0 \ psi \ (CSI = 0.71) \ ? \ \textbf{OK}$

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2.) Shear:

Members subject to shear stresses shall be proportioned so that the actual shear stress parallel to grain or shear force at any cross section of the bending member shall not exceed the adjusted shear design value:

$$f_V \leq F_V'$$
 (NDS Sec. 3.4.1)

where:

$$f_v =$$

 $F_v' = F_v(C_D)(C_M)(C_t)$

 $F_{vx'} = (265)(1.33)(1)(1) = 352.45 \text{ psi}$

Shear Reduction: For beams supported by full bearing on one surface and loads applied to the opposite surface, uniformly distributed loads within a distance, d, from supports equal to the depth of the bending member shall be pemitted to be ignored. For beams supported by full bearing on one surface and loads applied to the opposite surface, concentrated loads within a distance equal to the depth of the bending member shall be permitted to be multiplied by x/d where x is the distance from the beam support face to the load. See NDS 2015, Figure 3C.

 $f_v * = = 172.62 \text{ psi}$

 $f_v^* = 172.62 \text{ psi} < F_{vx'} = 352.45 \text{ psi} (CSI = 0.49)$? **OK**

No Reduction in Shear (conservative):

 $f_v = = 212.45 \text{ psi}$

 $f_v = 212.45 \text{ psi} < F_{vx'} = 352.45 \text{ psi} (CSI = 0.60)$? **OK**

3.) Deflection:

Bending deflections calculated per standard method of engineering mechanics for live load and total load:

LL Allowable: L/360 TL Allowable: L/240

 $E_x' = E_x(C_M)(C_t) = 1800000(1)(1) = 1800000 \text{ psi}$

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 $\Delta_{LL}=$ = 0.28 in.

 $(L/d)_{LL} = 176.00 / 0.28 = 633$

 $\Delta_{LL} = 0.28~in = L/633 < L/360$? OK

 $\Delta TL = = 0.49$ in.

 $(L/d)_{TL} = 176.00 / 0.49 = 357$

 $\Delta_{TL} = 0.49 \text{ in} = L/357 < L/240$? **OK**

4.) Bearing:

Members subject to bearing stresses perpendicular to the grain shall be proportioned so that the actual compressive stress perpendicular to grain shall be based on the net bearing area and shall not exceed the adjusted compression design value perpendicular to grain:

 $f_{c\,\perp} \leq F_{c\,\perp}$ ' (NDS Sec. 3.10.2)

where:

 $f_{c\perp} =$

 $F_{c\perp}' = F_{c\perp}(C_M)(C_t)$

 $F_{c \perp x}' = (650)(1)(1) = 650.00 \text{ psi}$

$$A_b = b x l_b = 2.5 x 4 = 10.00 in^2$$

 $f_{c\perp} = = 597.5 \text{ psi}$

 $f_{c\,\perp} = 597.5 \; psi < F_{c\,\perp\,x'} = 650.00 \; psi \; (CSI = 0.92) \; ?$ OK

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