



Beam Design

1. Beam Data

Load Type: Single Point Load
 Support: Simple Beam
 Beam Type: Glulam
 Species: Southern Pine
 Grade: 24F-V3 1.8E SP/SP
 Size: 3.5 x 9.625
 Design Span (L): 14.75 ft.
 Clear Span: 14.50 ft.
 Total Span: 15.00 ft.
 Bearing (lb): 3 in.
 Quantity (N): 1

2. Design Loads

Live Load: 1000 lbs
 Dead Load: 800 lbs
 Selfweight: 127.3 lbs
 Dist. Selfweight: 8.63 plf
 Total Weight: 129.4 lbs

3. Design Options

Lateral Support: 4' o/c
 Defl. Limits: 360/240
 Load Duration: 1.15
 Exposure: dry
 Temperature: $T \leq 100^{\circ}\text{F}$
 Orientation: Vertical

4. Design Assumptions and Notes

Code Standard: IBC 2015, NDS 2015
 Bending Stress: Parallel to Grain
 Notes:

5. Adjustment Factors

Factor	Description	F_b	F_t	F_v	F_c	$F_{c \perp}$	E/E_{min}
C_D	Load Duration Factor	1.15	1.15	1.15	1.15	-	-
C_M	Wet Service Factor	1	1	1	1	1	1
C_t	Temperature Factor	1	1	1	1	1	1
C_L	Beam Stability Factor	0.987	-	-	-	-	-
C_V	Volume Factor	1.0^b	-	-	-	-	-
C_{fu}	Flat Use Factor	N/A^c	-	-	-	-	-

a) Adjustment factors per AWC NDS 2015 and NDS 2015 Supplement.

b) The volume factor, C_V , shall not apply simultaneously with the beam stability factor, C_L . The lesser factor shall apply.

c) Only applies when glulam beam is loaded in bending about the y-y axis.

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6. Beam Calculations

Determine reference design values, sectional properties and self weight of beam:

$$A = b \times d$$

Invalid Equation, $S_y = \frac{b^2 d}{6}$

Invalid Equation, $I_y = \frac{b^3 d}{12}$

where:

b = Breadth of rectangular beam in bending (in.)

d = Depth of rectangular beam in bending (in.)

A = Cross sectional area of beam (in.²)

S_x = Section modulus about the X-X axis (in.³)

S_y = Section modulus about the Y-Y axis (in.³)

I_x = Moment of inertia about the X-X axis (in.⁴)

I_y = Moment of inertia about the Y-Y axis (in.⁴)

$$b = 3.500 \text{ in.}$$

$$d = 9.625 \text{ in.}$$

$$A = 3.500 \times 9.625 = 33.69 \text{ in.}^2$$

$$S_x = (3.500)(9.625)^2/6 = 54.04 \text{ in.}^3$$

$$S_y = (3.500)^2(9.625)/6 = 19.65 \text{ in.}^3$$

$$I_x = (3.500)(9.625)^3/12 = 260.07 \text{ in.}^4$$

$$I_y = (3.500)^3(9.625)/12 = 34.39 \text{ in.}^4$$

Reference Design Values from Table 5A NDS Supplement (Reference Design Values for Structural Glue Laminated Softwood Timber Combinations).

Species & Grade	F _{bx+}	F _{bx-}	F _{c⊥x}	F _{vx}	E _x	E _{minx}	F _{by}	F _{c⊥y}	F _{vy}	E _y	E _{miny}	F _t	F _c	G
24F-V3 1.8E SP/SP	2400	2000	740	300	1800000	950000	1700	650	260	1600000	850000	1150	1650	0.55

The following formula shall be used to determine the density of wood (lbs/ft³). (NDS Supplement Sec. 3.1.3)

$$\rho_w = 62.4 \left[\frac{G}{1 + G(0.009)(m.c.)} \right] \left[1 + \frac{m.c.}{100} \right]$$

where:

ρ_w = Density of wood (lbs/ft³)

G = Specific gravity of wood (dimensionless)

m.c. = Moisture content of wood (percentile)

$$G = 0.55$$

$$m.c. = 16 \% \text{ (Max. moisture content at dry service conditions)}$$

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Invalid Equation = 36.89 lbs/ft³

Volume_{total} = N[A x (L + lb)] = 1 x [33.69 x (177.00 + 3)] x (12 in./ft.)³ = 3.51 ft³

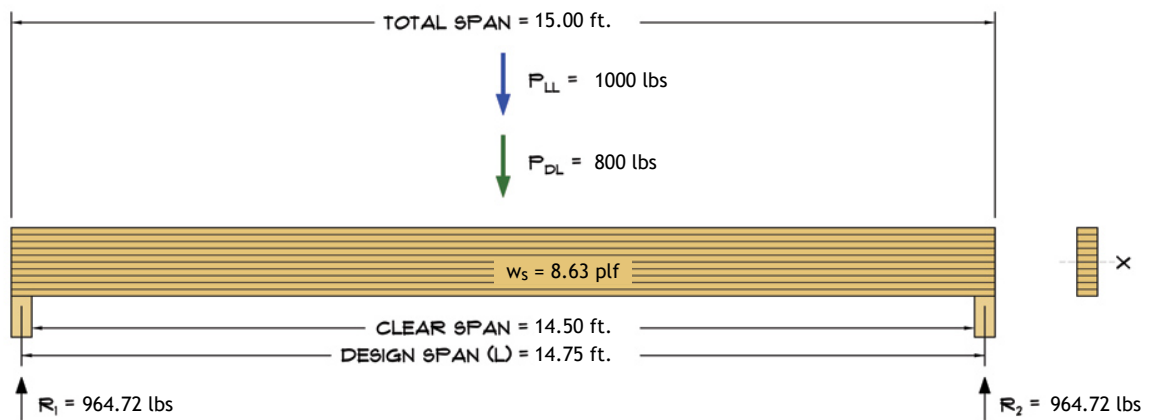
Volume_{span} = N[A x L] = 1 x [33.69 x 177.00] x (12 in./ft.)³ = 3.45 ft³

Total Weight (W_T) = ρ_w x Volume_{total} = 36.89 x 3.51 = 129.4 lbs

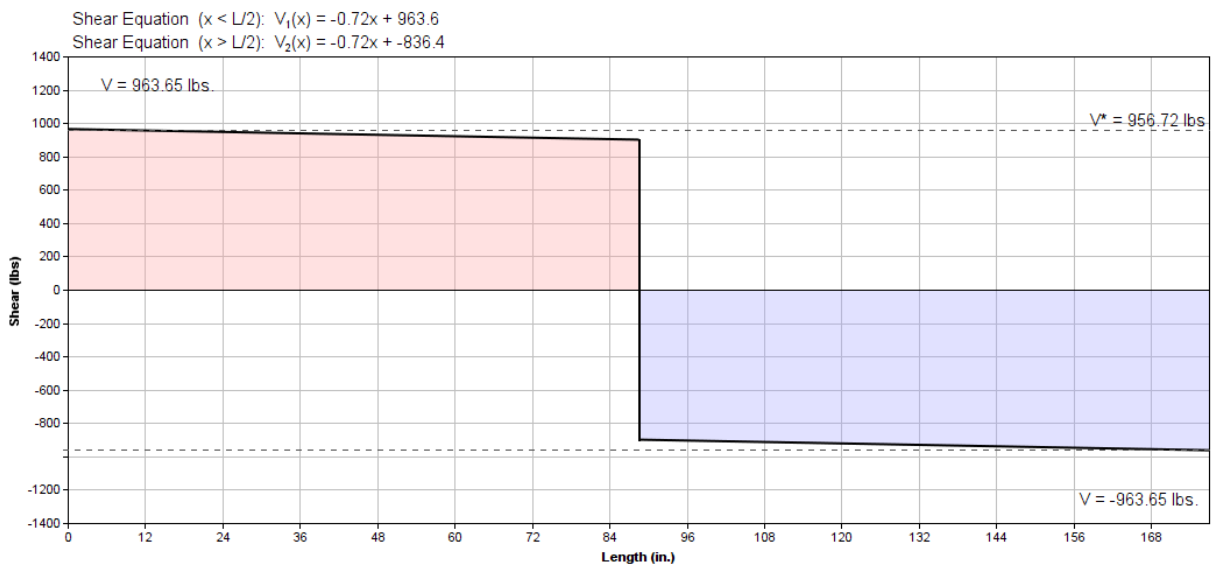
Self Weight (W_S) = ρ_w x Volume_{span} = 36.89 x 3.45 = 127.3 lbs

Distributed Self Weight (w_s) = **Invalid Equation** = 8.63 plf

Load, Shear and Moment Diagrams:

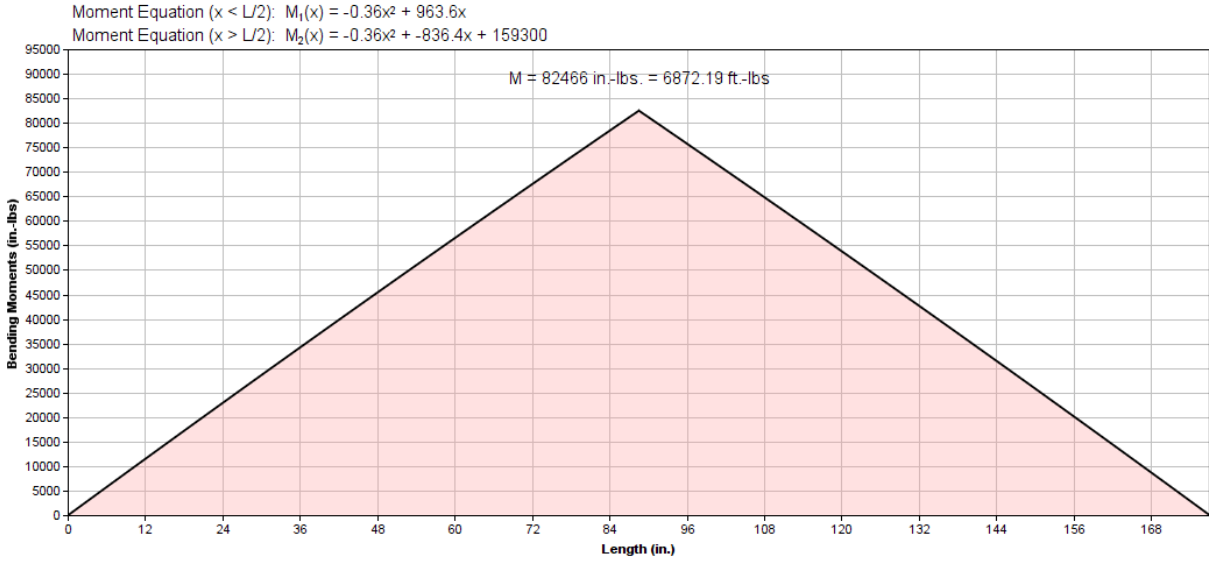


Beam - Shear Diagram



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Beam - Moment Diagram



1.) Bending:

Members subject to bending stresses shall be proportioned so that the actual bending stress or moment shall not exceed the adjusted bending design value:

$$f_b \leq F_b' \text{ (NDS Sec. 3.3.1)}$$

where:

$$f_b = M / S$$

$$F_{bx}' = F_{bx}(C_D)(C_M)(C_t)(C_V) \text{ or } F_{bx}' = F_{bx}(C_D)(C_M)(C_t)(C_L)$$

Beam is unbraced along its compression edge, lateral stability is considered below:

Slenderness Ratio for bending member RB:

$$l_u = \text{Unbraced Length} = 4 \text{ ft.}$$

$$l_u/d = \text{Invalid Equation} = 4.99$$

$$l_e = 2.06l_u = 2.06(48.0) = 98.88 \text{ in.} = 8.24 \text{ ft. (NDS Table 3.3.3)}$$

$$R_b = \text{Invalid Equation} = 8.81$$

$$R_b = 8.81 < 50 \text{ ? OK}$$

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Euler-based ASD critical buckling value for bending members:

$$E_{\min y}' = E_{\min y}(C_M)(C_t) = 850000(1)(1) = 850000 \text{ psi}$$

$$F_{bE} = \text{! Invalid Equation} = 13128.86 \text{ psi}$$

$$F_{bx}^* = F_{bx}(C_D)(C_M)(C_t) = (2400)(1.15)(1)(1) = 2760.00 \text{ psi}$$

Beam stability factor:

$$C_L = \text{! Invalid Equation} \text{! Invalid Equation} = 0.987$$

$$C_V = \text{! Invalid Equation} = 1.0$$

Lateral stability governs over volume effect:

$$C_L = 0.987 < C_V = 1.0$$

$$F_{bx}' = (2400)(1.15)(1)(0.987) = 2724.3 \text{ psi}$$

$$f_b = \text{! Invalid Equation} = 1526.0 \text{ psi}$$

$$f_b = 1526.0 \text{ psi} < F_{bx}' = 2724.3 \text{ psi} \text{ (CSI} = 0.56) \text{ ? OK}$$

2.) Shear:

Members subject to shear stresses shall be proportioned so that the actual shear stress parallel to grain or shear force at any cross section of the bending member shall not exceed the adjusted shear design value:

$$f_v \leq F_v' \text{ (NDS Sec. 3.4.1)}$$

where:

$$f_v = \frac{3V}{2A}$$

$$F_v' = F_v(C_D)(C_M)(C_t)$$

$$F_{vx}' = (300)(1.15)(1)(1) = 345.00 \text{ psi}$$

Shear Reduction: Uniformly distributed loads within a distance, d , from supports equal to the depth of the bending member shall be permitted to be ignored. Concentrated loads within a distance equal to the depth of the bending member from supports shall be permitted to be multiplied by x/d where x is the distance from the beam support face to the load. See NDS 2015, Figure 3C.

$$f_v^* = \text{! Invalid Equation} = 42.60 \text{ psi}$$

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$$f_v^* = 42.60 \text{ psi} < F_{vx}' = 345.00 \text{ psi} \text{ (CSI} = 0.12) \text{ ? OK}$$

No Reduction in Shear (conservative):

$$f_v = \text{! Invalid Equation} = 42.91 \text{ psi}$$

$$f_v = 42.91 \text{ psi} < F_{vx}' = 345.00 \text{ psi} \text{ (CSI} = 0.12) \text{ ? OK}$$

3.) Deflection:

Bending deflections calculated per standard method of engineering mechanics for live load and total load:

LL Allowable: L/360

TL Allowable: L/240

$$E_x' = E_x(C_M)(C_t) = 1800000(1)(1) = 1800000 \text{ psi}$$

$$\Delta_{LL} = \frac{P_{LL}L^3}{48E_x'(N \times I_x)} = \text{! Invalid Equation} = 0.25 \text{ in.}$$

$$(L/d)_{LL} = 177.00 / 0.25 = 717$$

$$\Delta_{LL} = 0.25 \text{ in} = L/717 < L/360 \text{ ? OK}$$

$$\Delta_{TL} = \text{! Invalid Equation} = \text{! Invalid Equation} = 0.46 \text{ in.}$$

$$(L/d)_{TL} = 177.00 / 0.46 = 382$$

$$\Delta_{TL} = 0.46 \text{ in} = L/382 < L/240 \text{ ? OK}$$

4.) Bearing:

Members subject to bearing stresses perpendicular to the grain shall be proportioned so that the actual compressive stress perpendicular to grain shall be based on the net bearing area and shall not exceed the adjusted compression design value perpendicular to grain:

$$f_{c\perp} \leq F_{c\perp}' \text{ (NDS Sec. 3.10.2)}$$

where:

$$f_{c\perp} = \frac{R}{A_b}$$

$$F_{c\perp}' = F_{c\perp}(C_M)(C_t)$$

$$F_{c\perp x}' = (740)(1)(1) = 740.00 \text{ psi}$$

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$$A_b = b \times l_b = 3.5 \times 3 = 10.50 \text{ in}^2$$

$$f_{c \perp} = \text{Invalid Equation} = 91.9 \text{ psi}$$

$$f_{c \perp} = 91.9 \text{ psi} < F_{c \perp x'} = 740.00 \text{ psi} \quad (\text{CSI} = 0.12) \quad ? \quad \text{OK}$$

*Disclaimer: The calculations produced herein are for initial design and estimating purposes only. The calculations and drawings presented do not constitute a fully engineered design. All of the potential load cases required to fully design an actual structure may not be provided by this calculator. For the design of an actual structure, a registered and licensed professional should be consulted as per IRC 2012 Sec. R802.10.2 and designed according to the minimum requirements of ASCE 7-10. The beam calculations provided by this online tool are for educational and illustrative purposes only. Medeek Design assumes no liability or loss for any designs presented and does not guarantee fitness for use.

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